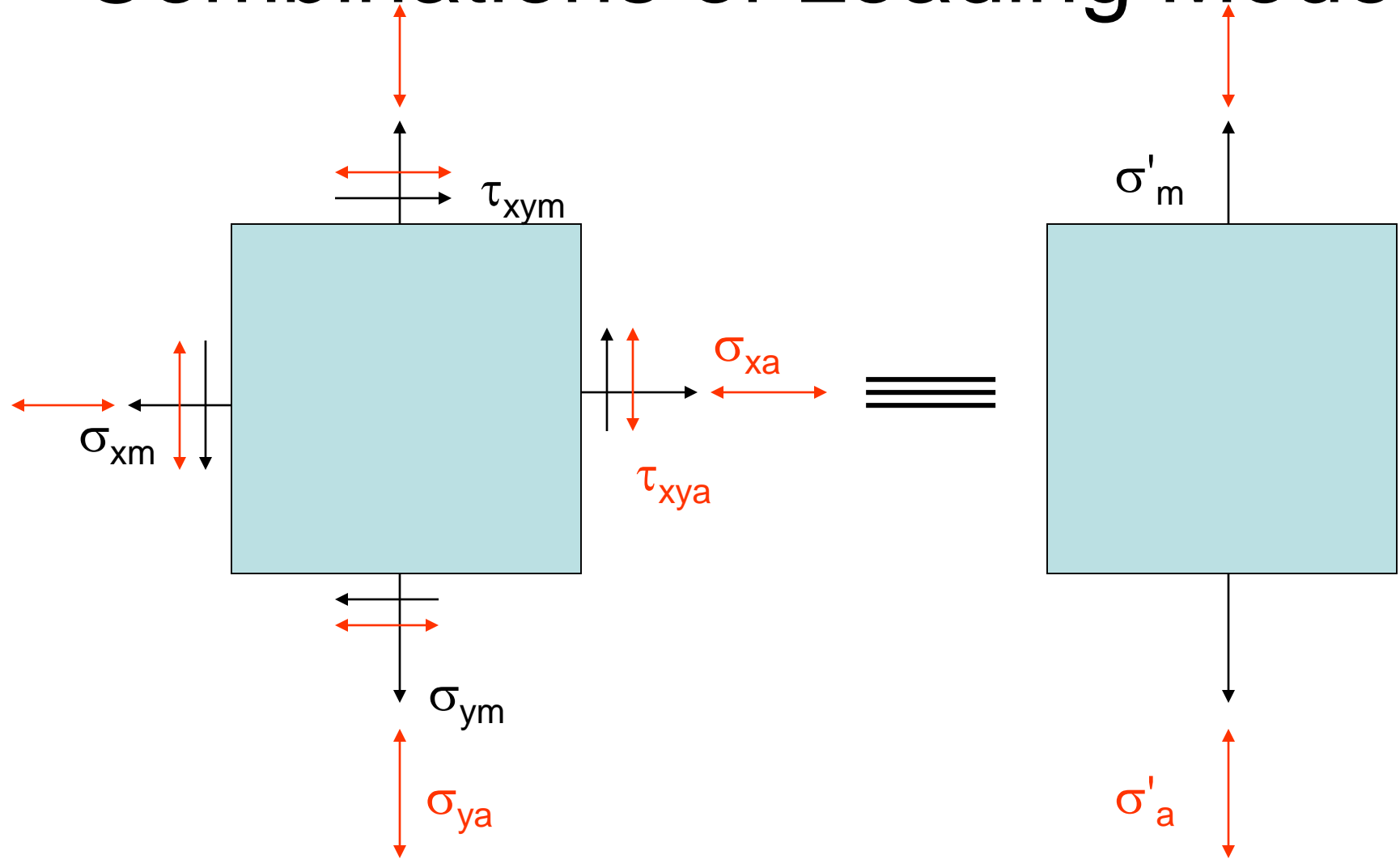


# Fatigue Failure II

# Combinations of Loading Modes

- In many cases stresses acting on machine elements are multiaxial stresses.
- Since most fatigue failures starts on the surface we consider biaxial case rather than full 3D.
- There are various approaches to deal with multiaxial fatigue. In this course, we adopt the simple approach of reducing the biaxial case to uniaxial case.

# Combinations of Loading Modes



# Combinations of Loading Modes

- $\sigma'_m$  and  $\sigma'_a$  are equivalent stresses. We use Von-Mises stress as the equivalent stress.
- $$\sigma'_m = (\sigma_{xm}^2 - \sigma_{xm}\sigma_{ym} + \sigma_{ym}^2 + 3\tau_{xym}^2)^{1/2} \quad (1)$$
- $$\sigma'_a = (\sigma_{xa}^2 - \sigma_{xa}\sigma_{ya} + \sigma_{ya}^2 + 3\tau_{xya}^2)^{1/2} \quad (2)$$
- If principal stresses are given, shear stress is taken as zero in the equations above.

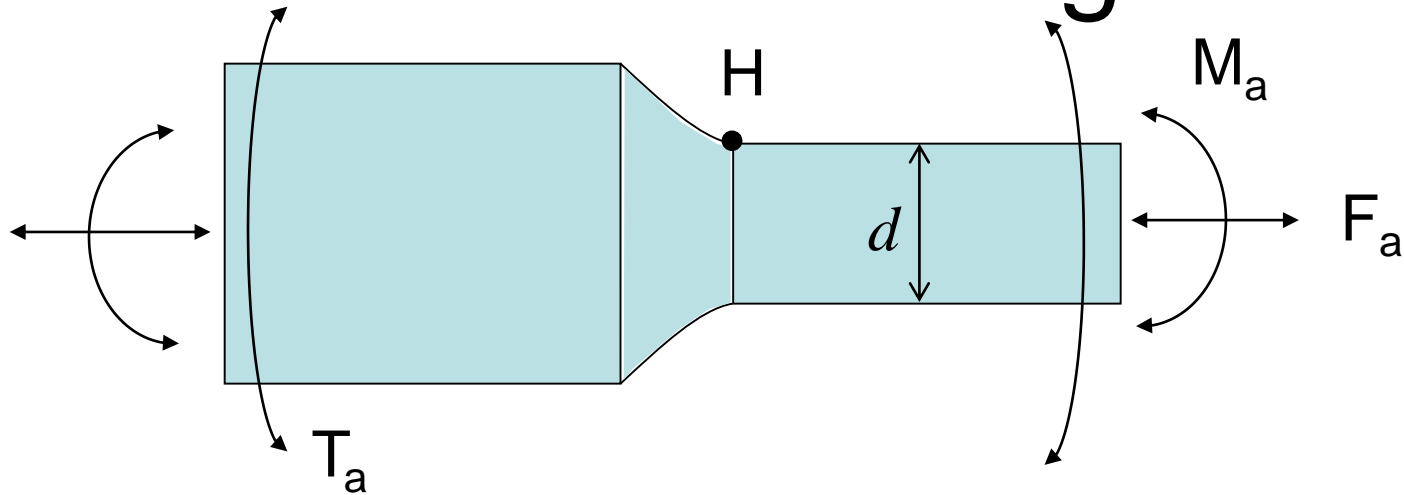
# Combinations of Loading Modes

- Once the equivalent stresses are obtained we may use Goodman or Soderberg approach as if we had uniaxial case.
- Note that in these equations, it is assumed that stresses are in phase, i.e. they reach their maximum and minimum values at the same time.

# Combinations of Loading Modes

- In biaxial case, we apply fatigue stress concentration factors  $K_f$  to alternating components of normal and/or shear stresses and then, take endurance limit modifying factor  $k_e=1$ .
- Similarly, we can use  $1/k_b$  to amplify stresses, rather than using  $k_b$  to reduce endurance limit.

# Combinations of Loading Modes



At point H which is the critical point, we have

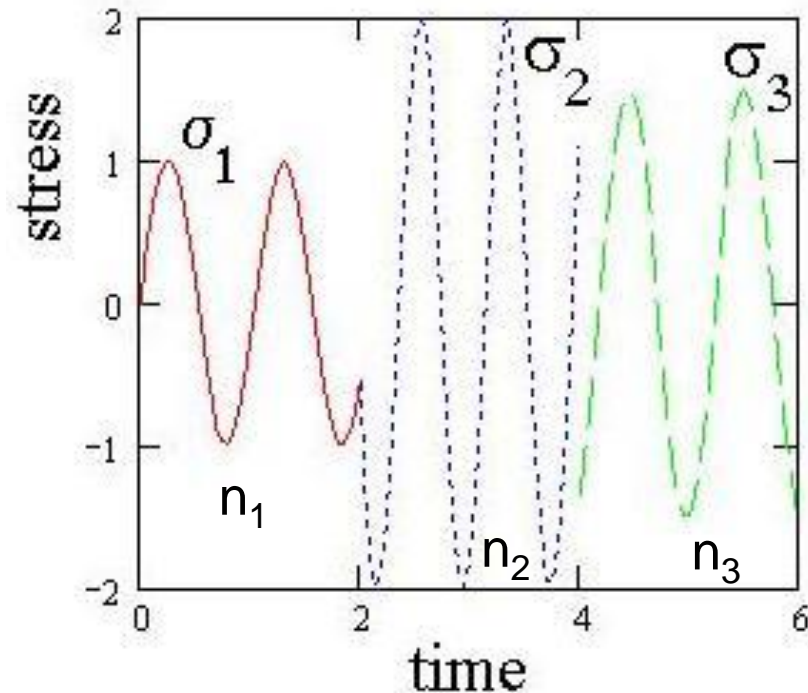
$$\sigma_{a,\text{axial}} = K_{f,\text{axial}} \frac{4F_a}{\pi d^2} \quad \sigma_{a,\text{bending}} = \frac{1}{k_b} K_{f,\text{bending}} \frac{32M_a}{\pi d^3}$$

$$\sigma_{xa} = \sigma_{a,\text{axial}} + \sigma_{a,\text{bending}} \quad \tau_{a,\text{torsion}} = \frac{1}{k_b} K_{f,\text{torsion}} \frac{16T_a}{\pi d^3}$$

Equations above should be used in eq.(2) to find equivalent alternating stress.

# Cumulative Fatigue Damage

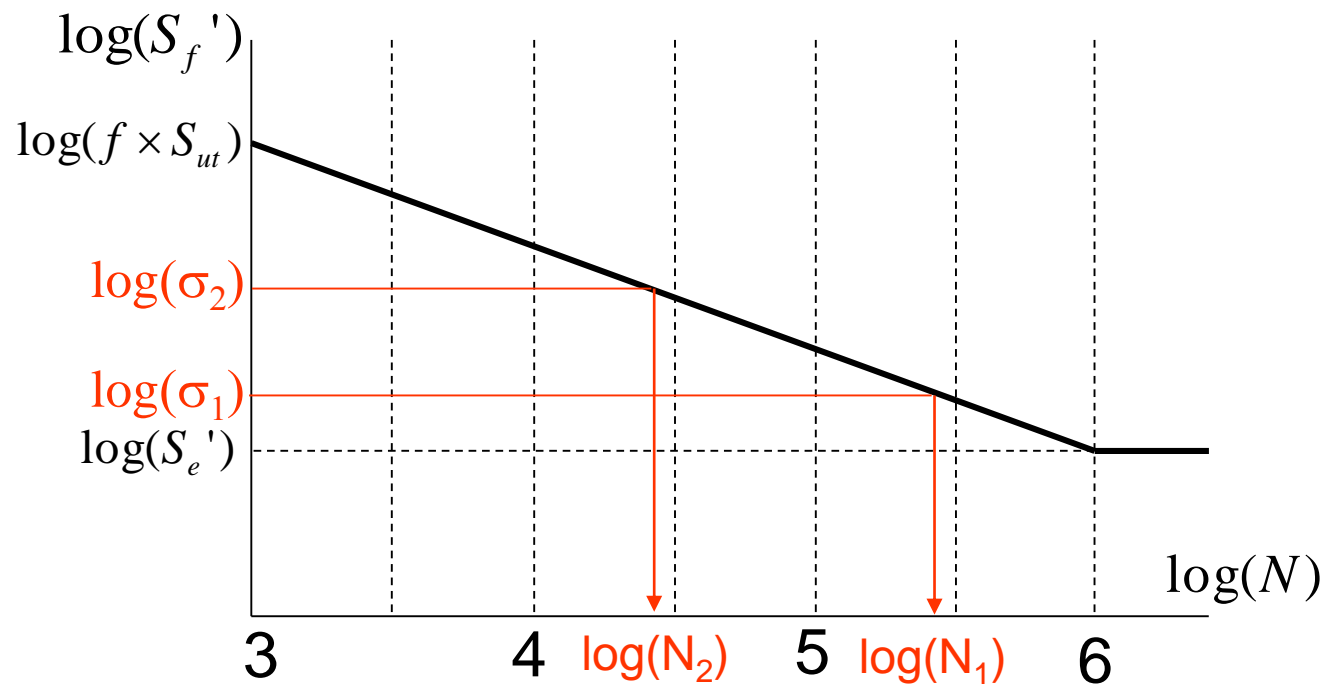
- If stress cycles with  $\sigma_{\max} > S_e$  are applied, fatigue life (N) for another stress level and endurance limit decreases. To predict this reduction in life there are two approaches.
  - Miner's Rule
  - Manson's Rule





# Cumulative Fatigue Damage

- stress  $\sigma_i$  is applied  $n_i$  cycles to the specimen,  $N_i$  is the life corresponding to stress  $\sigma_i$ .



# Miner's Rule

- According to Miner's Rule;

$$\sum \frac{n_i}{N_i} = \frac{n_1}{N_1} + \frac{n_2}{N_2} + \dots + \frac{n_m}{N_m} = C \quad 0.7 \leq C \leq 2.2$$

- $C$  is an experimentally determined constant.
- Unless otherwise is specified we take  $C=1$

# Miner's Rule

- Miner's Rule have deficiencies
  - It does not account for the order in which the stresses are applied, hence ignores any stress less than the initial endurance limit.
  - If an attempt is made to correct endurance limit by using Miner's rule, than  $S_{ut}$  will also become lowered, but this is not verified by the experiments.

# Manson's Rule

- Manson's rule does not have the deficiencies of Miner's Rule
- In Manson's rule, S-N diagram is updated in the same historical order with the applied stresses.
- $S_{ut}$  is kept constant and the new S-N line is drawn to pass through the point given by applied stress and remaining life.

# Manson's Rule

